### Modeling and Numerical Simulations of Active and Passive Forces Using Immersed Boundary Method

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# Abstract

This thesis uses the Immersed Boundary Method (IBM) to simulate the movement of a human heart. The IBM was developed by Charles Peskin in the 70's to solve Fluid-Structure Interaction models (FSI). The heart is embedded inside a fluid (blood) which moves according to the Navier-Stokes equation. The Navier-Stokes equation is solved by the Spectral Method. Forces on the heart muscle can be divided into two kinds: Active Force and Passive Force. Passive includes the effect of curvature (Peskin's model), spring model, and the torsional spring (or beam) model. Active force is modeled by the 3-element Hill model, which was used in the 30's to model skeletal muscle. We performed simulations with different combinations of these four forces. Numerical simulations are performed using MATLAB. We downloaded Peskin's code from the Internet and modified the Force.m file to include the above four forces. This thesis only considers heart muscle movement in the organ (macroscopic) scale.

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# Chapter 1

# Introduction

#### 1.1 The Left Ventricle

The heart chambers are separated by a wall of tissue called the septum. Each side has a small heart chamber called the atrium, which leads into a large pumping chamber called a ventricle. The heart has four heart chambers. The atria are the upper two chambers, and the ventricles are the lower two chambers.

The heart is about the size of the clenched fist. It lies in the front and middle of the chest, behind and slightly to the left of the breastbone. It is an organ that pumps blood to all parts of your body to provide it with the oxygen and nutrients in needs to function.

The left ventricle is one of the four chambers of the heart. It is located in the left portion of the heart below the left atrium, separated by the mitral valve. The wall of the chamber is about 9 to 12 mm thick, and it is 2 to 3 times size larger than right ventricle. In order to adapt to the function of the left ventricle, the chamber has a conical shape with a bottom upward and a sharp left to the front. The left ventricular cavity is divided into two parts, the inflow channel and the outflow channel, by the anterior flap of the mitral valve. As the heart contracts, blood eventually flows back into the left atrium, and then through the mitral valve, whereupon it next enters the left ventricle. Blood is pumped through the chambers, aided by four heart valves. The valves open and close to let the blood flow in only one direction. The physiological load on the ventricles requiring pumping of blood throughout the body and lungs is much greater than the pressure generated by the atria to fill the ventricles. Further, the left ventricle has thicker walls than the right because it needs to pump blood to most of the body while the right ventricle fills only the lungs. The normal blood flow is a cycle that flows like this; body-heart-lungs-heart-body.

#### **1.2** Navier-Stokes Equations

Navier–Stokes equations are useful because they describe the physics of many phenomena of scientific and engineering interest. They may be used to model the weather, ocean currents, water flow in a pipe and air flow around a wing. The Navier–Stokes equations, in their full and simplified forms, help with the design of aircraft and cars, the study of blood flow, the design of power stations, the analysis of pollution, and many other things. Coupled with Maxwell's equations, they can be used to model and study magnetohydrodynamics.

Navier-Stokes equations is one of the most important explanation to calculate fluid spread and velocity influenced by the force.

In physics, the Navier–Stokes equations, named after Claude-Louis Navier and George Gabriel Stokes, describe the motion of viscous fluid substances.

These balance equations arise from applying Isaac Newton's second law to fluid motion, together with the assumption that the stress in the fluid is the sum of a diffusing viscous term (proportional to the gradient of velocity) and a pressure term.

The Navier–Stokes equations are also of great interest in a purely mathematical sense. Despite their wide range of practical uses, it has not yet been proven whether solutions always exist in three dimensions and, if they do exist, whether they are smooth

The Navier–Stokes equations are strictly a statement of the balance of momentum. To fully describe fluid flow, more information is needed, how much depending on the assumptions made. This additional information may include boundary data, conservation of mass, balance of energy, and/or an equation of state. The Naiver-Stokes equations are given as following:

$$\rho\left(\frac{\partial \mathbf{u}(x,t)}{\partial t} + \mathbf{u}(x,t) \cdot \nabla \mathbf{u}(x,t)\right) = -\nabla p(x,t) + \mu \Delta \mathbf{u}(x,t) + \mathbf{f}(x,t)$$

For this case, in order to keep the fluid incompressible, we assume

$$\nabla \cdot \mathbf{u}(x,t) = 0$$

The Navier–Stokes equations, even when written explicitly for specific fluids, are rather generic in nature and their proper application to specific problems can be very diverse. This is partly because there is an enormous variety of problems that may be modeled, ranging from as simple as the distribution of static pressure to as complicated as multiphase flow driven by surface tension.

In this thesis, the Navier-Stokes equation will be needed to model blood flow in the heart.

#### **1.3** Immersed Boundary Methods

Immersed boundary methods were first introduced by Peskin (1972) [2] to simulate cardiac mechanics and associated blood flow. For this topic, the following materials we cited the knowledge from Jiyuan Tu's book directly [10, 11]. One distinguishing feature of this approach is the ability to perform the entire simulation on a fixed Cartesian grid. A novel procedure was developed by directly imposing the effect of the immersed boundary on the flow; the requirement for the grids to conform to the complex geometrical structure of the heart was thus avoided. Since its inception, numerous modifications and refinements have been proposed, and a number of variants of this approach currently exist.

When applying an immersed boundary method to solve for fluid flow in a complex geometry, we select a (usually) rectangular domain that contains both the fluid region and also the bounding solid. The resulting rectangular domain now contains fluid regions where the fluid velocity is governed by the Navier-Stokes equations and solid regions where the velocity is zero, or given. The domain is resolved by a regular structured grid and some grid points (or control volumes) are in the fluid region and some are in the solid region.

In the 1970s, Peskin developed the immersed-boundary method to simulate flexible membranes in fluid flows. The membrane-fluid interaction is accomplished by distributing membrane forces as local fluid forces and updating membrane configuration according to local flow velocity. Since then, the immersed-boundary method has been widely employed to study various situations, including cell deformation in micropipettes, leukocyte adhesion and movement, multiphase flows, red blood cells deformation and aggregation in shear flows and the behavior of biofilms.

In immersed-boundary method, the key idea is that the membrane force  $f(x_m)$  at a membrane marker  $x_m$  induced by membrane deformation is distributed to the nearby fluid grid points  $x_f$  by [3]:

$$f(x_f) = \sum_{x_m} D(x_f - x_m) F(x_m)$$
(1.1)

through a discrete delta function D(x), which is chosen to approximate the properties of the Dirac delta function.

# Chapter 2

### **Immersed Boundary Method**

The material in this section comes from Peskin's paper 2002 [3]. In this chapter, we will describe some numerical methods which is essential to help us understand and solve the Navier-Stokes equation in 2D.

In the study, we apply the immersed boundary method developed by Charles Peskin. The fluid is represented on the Eulerian grid, and the structure is represented on the Lagrangian coordinate.

Recall the Navier-Stokes equations to solve our fluid motion in two dimensions is

$$\rho\left(\frac{\partial \mathbf{u}(x,t)}{\partial t} + \mathbf{u}(x,t) \cdot \nabla \mathbf{u}(x,t)\right) = -\nabla p(x,t) + \mu \Delta \mathbf{u}(x,t) + \mathbf{f}(x,t)$$
$$\nabla \cdot \mathbf{u}(x,t) = 0$$

where  $\mathbf{u}(x,t) = (u(x,t), v(x,t))$  is the fluid velocity, p(x,t) is the pressure, and  $\mathbf{f}(x,t)$  is the force per unit volume (area in 2D) applied to the fluid by the immersed boundary.

#### 2.1 Notation

Let 
$$\mathbf{u} = \begin{pmatrix} u \\ v \end{pmatrix}$$

The nonlinear term in the Navier-Stokes equation in 2D is,

$$\mathbf{u} \cdot \nabla \mathbf{u} = \begin{pmatrix} u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \\ u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} \end{pmatrix}$$

We use the notation that  $D_h^0$  is the vector difference operator. Thus,  $D_h^0 \phi$  is the central difference approximation to the gradient of  $\phi$ , and  $D_h^0 \cdot u$  is the central difference approximation

to the divergence of u. Let,

$$D\alpha = \begin{pmatrix} \frac{\partial \alpha}{\partial x} \\ \frac{\partial \alpha}{\partial y} \end{pmatrix}$$

Therefore, we have

$$S(\mathbf{u})\alpha = \frac{1}{2}\mathbf{u} \cdot \mathbf{D}\alpha + \frac{1}{2}\mathbf{D} \cdot (\mathbf{u}\alpha)$$

$$= \frac{1}{2}\left(u\frac{\partial\alpha}{\partial x} + v\frac{\partial\alpha}{\partial y}\right) + \frac{1}{2}D \cdot \begin{pmatrix}u\alpha\\v\alpha\end{pmatrix}$$

$$= \frac{1}{2}\left(u\frac{\partial\alpha}{\partial x} + v\frac{\partial\alpha}{\partial y}\right) + \frac{1}{2}\left((u\alpha)_{x} + (v\alpha)_{y}\right)$$

$$= \frac{1}{2}\left(u\frac{\partial\alpha}{\partial x} + v\frac{\partial\alpha}{\partial y}\right) + \frac{1}{2}\left(u_{x}\alpha + u\alpha_{x} + v_{y}\alpha + v\alpha_{y}\right)$$

$$= u\frac{\partial\alpha}{\partial x} + v\frac{\partial\alpha}{\partial y} + \frac{1}{2}\left(u_{x} + v_{y}\right)\alpha$$

$$= u\frac{\partial\alpha}{\partial x} + v\frac{\partial\alpha}{\partial y} \qquad (2.1)$$

because  $u_x + v_y = 0$ , which follows from the incoompressibility condition  $\nabla \cdot \mathbf{u} = \mathbf{0}$ 

#### 2.2 Numerical Method

The temporal discretization that we currently use (Lai and Peskin 2000, McQueen and Peskin 2001) [8, 9] is a second-order Runge–Kutta method, based primarily on the midpoint rule. For a system of ordinary differential equations of the for

$$\frac{dy}{dt} = f(y)$$

such a scheme looks like this:

$$\frac{y^{n+\frac{1}{2}} - y^n}{\Delta t/2} = f(y^n)$$
(2.2)

$$\frac{y^{n+1} - y^n}{\Delta t} = f(y^{n+\frac{1}{2}}) \tag{2.3}$$

where the superscript is the time-step index. The salient feature of this scheme is that each time-step involves a 'preliminary substep' to the halftime level followed by a 'final sub-step' from time level n to n + 1, in which the results of the preliminary substep are used. The preliminary sub-step involves a first-order accurate scheme (forward Euler in the above example), and the final substep is done by a second-order accurate scheme (here, the midpoint rule). One would think that the accuracy would be limited by the least accurate sub-step, but the magic of Runge–Kutta is that the overall scheme is second-order accurate.

#### 2.3 First-Half Step

The following formula are taken from page 507 of Peskin (2002) [3]. Consider the half step and integrate the Navier–Stokes equations on the Eulerian grid  $g_h$  from time level n to time level  $n + \frac{1}{2}$ .

$$\rho^{n+\frac{1}{2}} \left( \frac{\mathbf{u}^{n+\frac{1}{2}} - \mathbf{u}^n}{\Delta t/2} + S_h(\mathbf{u}^n) \mathbf{u}^n \right) + \mathbf{D}_h^0 \, \tilde{p}^{n+\frac{1}{2}} = \mu L_h \mathbf{u}^{n+\frac{1}{2}} + \mathbf{f}^{n+\frac{1}{2}}$$
(2.4)  
$$\mathbf{D}_h^0 \cdot \mathbf{u}^{n+\frac{1}{2}} = 0$$

From the above definition, since  $\mathbf{u}^{\mathbf{n}} = \begin{pmatrix} u^n \\ v^n \end{pmatrix}$ From (2.1),

$$S_h(\mathbf{u}^n)\,\mathbf{u}^n = \begin{pmatrix} u^n u^n_x + v^n u^n_y \\ u^n v^n_x + v^n v^n_y \end{pmatrix}$$

From the first component of (2.4), we have

$$u^{n+\frac{1}{2}} = u^n + \frac{\Delta t/2}{\rho^{n+\frac{1}{2}}} \left( f^{n+\frac{1}{2}} - \rho^{n+\frac{1}{2}} (u^n u^n_x + v^n u^n_y) \right)$$
(2.5)

Similarity, we have  $v^{n+\frac{1}{2}}$ 

$$v^{n+\frac{1}{2}} = v^n + \frac{\Delta t/2}{\rho^{n+\frac{1}{2}}} \left( f^{n+\frac{1}{2}} - \rho^{n+\frac{1}{2}} (u^n v_x^n + v^n v_y^n) \right)$$
(2.6)

Note that this is the formula without pressure and Laplacian terms.

#### 2.4 Spectral Method

The spectral method comes from Trefethen's book [7]. The Fast Fourier Transform of the function p evaluated at the grid-point  $(mh_x, nh_y)$  is

$$\hat{p} = \sum_{j,k} p_{j,k} e^{imh_x 2\pi j + inh_y 2\pi k}$$

Using central difference, the second partial derivative of p at the same grid-point is given by

$$\hat{p}_{xx} = \sum_{j,k} \frac{1}{h_x^2} \left( e^{ih_x 2\pi j} + e^{-ih_x 2\pi j} - 2 \right) e^{imh_x 2\pi j + inh_y 2\pi k} p_{j,k}$$

Let  $\theta = h_x 2\pi j$ , then above becomes

$$\hat{p}_{xx} = \sum_{j,k} \frac{1}{h_x^2} (e^{i\theta} + e^{-i\theta} - 2) e^{imh_x 2\pi j + inh_y 2\pi k} p_{j,k}$$

Since,

$$e^{i\theta} + e^{-i\theta} = \cos\theta + i\sin\theta + \cos(-\theta) + i\sin(-\theta)$$
  
=  $2\cos\theta$   
 $2\cos\theta - 2 = 2\left(1 - 2\sin^2\left(\frac{\theta}{2}\right)\right) - 2$   
=  $-4\sin^2\left(\frac{\theta}{2}\right)$ 

Then,

$$\hat{p}_{xx} = \sum_{j,k} \frac{1}{h_x^2} \left( -4\sin^2\left(\frac{h_x 2\pi j}{2}\right) \right) e^{imh_x 2\pi j + inh_y 2\pi k} p_{j,k}$$
(2.7)  
$$= -\sum_{j,k} e^{imh_x 2\pi j + inh_y 2\pi k} p_{j,k} \frac{\sin^2\left(\frac{h_x 2\pi j}{2}\right)}{\left(\frac{h_x}{2}\right)^2}$$
$$= -\sum_{j,k} e^{imh_x 2\pi j + inh_y 2\pi k} p_{j,k} \left(\frac{\sin(dx \ 2\pi j)}{dx}\right)^2$$
(2.8)

where  $dx = \frac{h_x}{2}$ . For the first derivative,

$$\hat{p}_{x} = \sum_{j,k} \frac{1}{2h_{x}} \left( e^{ih_{x}2\pi j} - e^{-ih_{x}2\pi j} \right) e^{imh_{x}2\pi j + inh_{y}2\pi k} p_{j,k}$$

$$= \sum_{j,k} \frac{1}{2h_{x}} 2i \sin(h_{x}2\pi j) e^{imh_{x}2\pi j + inh_{y}2\pi k} p_{j,k}$$

$$= i \sum_{j,k} \frac{\sin(h_{x}2\pi j)}{h_{x}} e^{imh_{x}2\pi j + inh_{y}2\pi k} p_{j,k}$$
(2.9)

In Fourier space, (2.4) becomes

$$\rho^{n+\frac{1}{2}} \left( \frac{\hat{\mathbf{u}}^{n+\frac{1}{2}} - \hat{\mathbf{u}}^n}{\Delta t/2} + \widehat{S_h(\mathbf{u}^n)\mathbf{u}^n} \right) = -\nabla \hat{p}^{n+\frac{1}{2}} + \hat{\mathbf{f}}^{n+\frac{1}{2}} + \mu L_h \hat{\mathbf{u}}^{n+\frac{1}{2}} \nabla \cdot \hat{\mathbf{u}}^{n+\frac{1}{2}} = 0$$

Rearranging,

$$\begin{pmatrix} \frac{\rho^{n+\frac{1}{2}}}{\Delta t/2} - \mu L_h \end{pmatrix} \hat{\mathbf{u}}^{n+\frac{1}{2}} = -\nabla \hat{p}^{n+\frac{1}{2}} + \frac{\rho^{n+\frac{1}{2}}}{\Delta t/2} \left( \hat{\mathbf{u}}^n + \frac{\Delta t/2}{\rho^{n+\frac{1}{2}}} \, \hat{\mathbf{f}}^{n+\frac{1}{2}} - \frac{\Delta t}{2} \, \widehat{S_h(\mathbf{u}^n)\mathbf{u}^n} \right) \\ \nabla \cdot \left( \frac{\rho^{n+\frac{1}{2}}}{\Delta t/2} - \mu L_h \right) \hat{\mathbf{u}}^{n+\frac{1}{2}} = -\Delta \hat{p}^{n+\frac{1}{2}} + \nabla \cdot \frac{\rho^{n+\frac{1}{2}}}{\Delta t/2} \left( \hat{\mathbf{u}}^n + \frac{\Delta t/2}{\rho^{n+\frac{1}{2}}} \, \hat{\mathbf{f}}^{n+\frac{1}{2}} - \frac{\Delta t}{2} \, \widehat{S_h(\mathbf{u}^n)\mathbf{u}^n} \right) \\ \Delta \hat{p}^{n+\frac{1}{2}} = \frac{\rho^{n+\frac{1}{2}}}{\Delta t/2} \left( (\widehat{rhs_u})_x + (\widehat{rhs_v})_y \right) \tag{2.10}$$

The left hand side is zero because of the incompressibility condition. The right side becomes,

$$\Delta \hat{p}^{n+\frac{1}{2}} = \frac{\rho^{n+\frac{1}{2}}}{\Delta t/2} \left( (\widehat{rhs}_u)_x + (\widehat{rhs}_u)_y \right)$$
(2.11)

where,

$$\binom{rhs_u}{rhs_v} = -\frac{\Delta t}{2} S_h(\mathbf{u}) \mathbf{u}^n + \frac{\Delta t/2}{\rho^{n+\frac{1}{2}}} \mathbf{f}^{n+\frac{1}{2}}$$
(2.12)

To simplified, we use  $\hat{p}$  substituting  $\hat{p}^{n+\frac{1}{2}}$  Since, from (2.8), we have

$$\Delta \hat{p} = \hat{p}_{xx} + \hat{p}_{yy}$$
  
$$\Delta \hat{p} = -\sum_{j,k} \left( \left( \frac{\sin(2\pi j \ dx)}{dx} \right)^2 + \left( \frac{\sin(2\pi k \ dy)}{dy} \right)^2 \right) e^{imh_x 2\pi j + inh_y 2\pi k} p_{j,k}$$

From above,

$$\Delta \hat{p}_{j,k}^{n+\frac{1}{2}} = -\left(\left(\frac{\sin(2\pi j \ dx)}{dx}\right)^2 + \left(\frac{\sin(2\pi k \ dy)}{dy}\right)^2\right) p_{j,k} \tag{2.13}$$

And from (2.9), we have

$$(\widehat{rhs_u})_x = i \sum_{j,k} \left(\frac{\sin(2\pi k \, dx)}{dx}\right) e^{imh_x 2\pi j + inh_y 2\pi k} rhs_u$$
$$(\widehat{rhs_v})_y = i \sum_{j,k} \left(\frac{\sin(2\pi j \, dy)}{dx}\right) e^{imh_x 2\pi j + inh_y 2\pi k} rhs_v$$

Equating (2.11) and (2.13), for each distinct  $p_{j,k}$  we have

$$-\left(\left(\frac{\sin(2\pi j\ dx)}{dx}\right)^2 + \left(\frac{\sin(2\pi k\ dy)}{dy}\right)^2\right)p_{j,k}^{n+\frac{1}{2}}$$
$$= \frac{\rho^{n+\frac{1}{2}}}{\Delta t/2}\left(i\left(\frac{\sin(2\pi j\ dx)}{dx}\right)rhs_u + i\left(\frac{\sin(2\pi k\ dy)}{dy}\right)rhs_v\right)$$

This formula uses to find  $\hat{p}^{n+\frac{1}{2}}$ 

### 2.5 Back to the First-Half Step

From the above section, we find the pressure term  $\hat{p}_{j,k}$ . We now solve for the fluid velocity term  $\mathbf{u}^{\mathbf{n}+\frac{1}{2}}$ . From (2.4),

$$\rho^{n+\frac{1}{2}} \left( \frac{\hat{\mathbf{u}}^{n+\frac{1}{2}} - \hat{\mathbf{u}}^{n}}{\Delta t/2} + \widehat{S_{h}(\mathbf{u}^{n})\mathbf{u}^{n}} \right) = -\nabla \hat{p}^{n+\frac{1}{2}} + \hat{\mathbf{f}}^{n+\frac{1}{2}} + \mu L_{h} \, \hat{\mathbf{u}}^{n+\frac{1}{2}} \\
\left( \frac{\rho^{n+\frac{1}{2}}}{\Delta t/2} - \mu L_{h} \right) \hat{\mathbf{u}}^{n+\frac{1}{2}} = -\nabla \hat{p}^{n+\frac{1}{2}} + \frac{\rho^{n+\frac{1}{2}}}{\Delta t/2} \left( \hat{\mathbf{u}}^{n} + \frac{\Delta t/2}{\rho^{n+\frac{1}{2}}} \, \hat{\mathbf{f}}^{n+\frac{1}{2}} - \frac{\Delta t}{2} \, \widehat{S_{h}(\mathbf{u}^{n})\mathbf{u}^{n}} \right) \\
= -\nabla \hat{p}^{n+\frac{1}{2}} + \frac{\rho^{n+\frac{1}{2}}}{\Delta t/2} \left( \hat{\mathbf{u}}^{n} + \left( \frac{\widehat{rhs_{u}}}{rhs_{v}} \right) \right)$$

We know that

$$\hat{\mathbf{u}}^{n+\frac{1}{2}} = \begin{pmatrix} \hat{u}^{n+\frac{1}{2}} \\ \hat{v}^{n+\frac{1}{2}} \end{pmatrix}$$
(2.14)

Thus, from (2.9) and (2.4), we have

$$\hat{u}^{n+\frac{1}{2}} = \frac{-\nabla \hat{p} + \frac{\rho^{n+\frac{1}{2}}}{\Delta t/2} \widehat{rhs_{u}}}{\left(\frac{\rho^{n+\frac{1}{2}}}{\Delta t/2} - \mu L_{h}\right)}$$
$$= \frac{\widehat{rhs_{u}} - i\left(\frac{\Delta t/2}{\rho^{n+\frac{1}{2}}}\right) \left(\frac{\sin(2\pi j \ dx)}{dx}\right) \hat{p}^{n+\frac{1}{2}}}{1 - \frac{\Delta t/2}{\rho^{n+\frac{1}{2}}} \ \mu L_{h}}$$
(2.15)

Similarly, to the second component of  $\mathbf{u}^{\mathbf{n}+\frac{1}{2}},$  we have

$$\hat{v}^{n+\frac{1}{2}} = \frac{-\nabla \hat{p} + \frac{\rho^{n+\frac{1}{2}}}{\Delta t/2} \widehat{rhs_{v}}}{\left(\frac{\rho^{n+\frac{1}{2}}}{\Delta t/2} - \mu L_{h}\right)}$$
$$= \frac{\widehat{rhs_{v}} - i\left(\frac{\Delta t/2}{\rho^{n+\frac{1}{2}}}\right) \left(\frac{\sin(2\pi k \, dy)}{dy}\right) \hat{p}^{n+\frac{1}{2}}}{1 - \frac{\Delta t/2}{\rho^{n+\frac{1}{2}}} \mu L_{h}}$$
(2.16)

We now find  $L_h$  in (2.15) and (2.16)

From (2.7), we have

$$\hat{\mathbf{u}}_{xx}^{n+\frac{1}{2}} = \sum_{j,k} \frac{1}{h_x^2} \left( -4\sin^2\left(\frac{h_x 2\pi j}{2}\right) \right) e^{imh_x 2\pi j + inh_y 2\pi k} \mathbf{u}_{j,k}^{n+\frac{1}{2}}$$
$$= -\sum_{j,k} e^{imh_x 2\pi j + inh_y 2\pi k} \frac{\sin^2\left(\frac{h_x 2\pi j}{2}\right)}{\left(\frac{h_x}{2}\right)^2} \mathbf{u}_{j,k}^{n+\frac{1}{2}}$$

Let  $dx = \frac{hx}{2}$  and  $dy = \frac{hy}{2}$ , then

$$L_h = -\left(\frac{\sin(2\pi j \ dx)}{dx}\right)^2 - \left(\frac{\sin(2\pi k \ dy)}{dy}\right)^2$$

The denominator of (2.15) and (2.16) is equal to

$$1 - \frac{\Delta t/2}{\rho^{n+\frac{1}{2}}} \,\mu L_h = 1 + \frac{\Delta t/2}{\rho^{n+\frac{1}{2}}} \mu \left( \left( \frac{\sin(2\pi j \, dx)}{dx} \right)^2 + \left( \frac{\sin(2\pi k \, dy)}{dy} \right)^2 \right)$$

Equations (2.15) and (2.16) allow us to find  $\hat{u}^{n+\frac{1}{2}}$  and  $\hat{v}^{n+\frac{1}{2}}$ , then apply the inverse Fourier series to obtain  $\begin{pmatrix} u^{n+\frac{1}{2}} \\ v^{n+\frac{1}{2}} \end{pmatrix}$ .

#### 2.6 Second-Half Step

Now we proceed directly to the Navier–Stokes equations. We are going to integrate the Navier–Stokes equations from time level n to the time level n + 1 using (2.3). We use a similar method in above section.

$$\rho^{n+\frac{1}{2}} \left( \frac{\mathbf{u}^{n+1} - \mathbf{u}^n}{\Delta t} + S_h(\mathbf{u}^{n+\frac{1}{2}}) \mathbf{u}^{n+\frac{1}{2}} \right) + \mathbf{D}_h^0 p^{n+\frac{1}{2}} = \mu L_h \left( \frac{\mathbf{u}^n + \mathbf{u}^{n+1}}{2} \right) + \mathbf{f}^{n+\frac{1}{2}} (2.17)$$
$$\mathbf{D}_h^0 \cdot \mathbf{u}^{n+1} = 0$$

Thus, we could find our  $S_h(\mathbf{u}^{n+\frac{1}{2}})\mathbf{u}^{n+\frac{1}{2}}$ , using the notation in section 2.1,

$$S_{h}(\mathbf{u}^{n+\frac{1}{2}}) u^{n+\frac{1}{2}} = \frac{1}{2} \mathbf{u}^{n+\frac{1}{2}} \cdot (D_{h}^{0} u^{n+\frac{1}{2}}) + \frac{1}{2} D_{h}^{0} \cdot (\mathbf{u}^{n+\frac{1}{2}} u^{n+\frac{1}{2}})$$
$$= u^{n+\frac{1}{2}} u_{x}^{n+\frac{1}{2}} + v^{n+\frac{1}{2}} u_{y}^{n+\frac{1}{2}}$$
(2.18)

$$S_{h}(\mathbf{u}^{n+\frac{1}{2}}) v^{n+\frac{1}{2}} = \frac{1}{2} \mathbf{u}^{n+\frac{1}{2}} \cdot (D_{h}^{0} v^{n+\frac{1}{2}}) + \frac{1}{2} D_{h}^{0} \cdot (\mathbf{u}^{n+\frac{1}{2}} v^{n+\frac{1}{2}})$$
$$= u^{n+\frac{1}{2}} v_{x}^{n+\frac{1}{2}} + v^{n+\frac{1}{2}} v_{y}^{n+\frac{1}{2}}$$
(2.19)

Therefore, we have  $S_h(\mathbf{u}^{n+\frac{1}{2}})\mathbf{u}^{n+\frac{1}{2}}$ ,

$$S_{h}(\mathbf{u}^{n+\frac{1}{2}})\mathbf{u}^{n+\frac{1}{2}} = \begin{pmatrix} u^{n+\frac{1}{2}}u_{x}^{n+\frac{1}{2}} + v^{n+\frac{1}{2}}u_{y}^{n+\frac{1}{2}} \\ u^{n+\frac{1}{2}}v_{x}^{n+\frac{1}{2}} + v^{n+\frac{1}{2}}v_{y}^{n+\frac{1}{2}} \end{pmatrix}$$
(2.20)

From (2.17), we have:

$$\left(1 - \frac{\Delta t}{\rho^{n+\frac{1}{2}}} \frac{\mu L_h}{2}\right)u^{n+1} = u^n + \frac{\Delta t}{\rho^{n+\frac{1}{2}}} \left(f^{n+\frac{1}{2}} - \mu L_h \frac{u^n}{2} - \rho^{n+\frac{1}{2}} u^{n+\frac{1}{2}} u^{n+\frac{1}{2}}_x - \rho^{n+\frac{1}{2}} v^{n+\frac{1}{2}} u^{n+\frac{1}{2}}_y u^{n+\frac{1}{2}}_y\right)$$

Note that this is the formula without pressure term. Apply for the fast Fourier Transform again.

Equation (2.17) in Fourier space is

$$\rho^{n+\frac{1}{2}} \left( \frac{\hat{\mathbf{u}}^{n+1} - \hat{\mathbf{u}}^n}{\Delta t} + \overline{S_h(\mathbf{u}^{n+\frac{1}{2}})\mathbf{u}^{n+\frac{1}{2}}} \right) = -\nabla \hat{p}^{n+\frac{1}{2}} + \mu L_h \left( \frac{\hat{\mathbf{u}}^n + \hat{\mathbf{u}}^{n+1}}{2} \right) + \hat{\mathbf{f}}^{n+\frac{1}{2}}$$
$$\mathbf{D}_h^0 \cdot \hat{\mathbf{u}}^{n+1} = 0$$

Thus, the (j,k) term in the Fourier series is

$$\left(\frac{\rho^{n+\frac{1}{2}}}{\Delta t} - \mu \frac{L_{h}}{2}\right) \hat{\mathbf{u}}^{n+1} = -\nabla \hat{p}^{n+\frac{1}{2}} \\
+ \frac{\rho^{n+\frac{1}{2}}}{\Delta t} \left(\hat{\mathbf{u}}^{n} + \frac{\Delta t}{\rho^{n+\frac{1}{2}}} \mu \frac{L_{h}}{2} \hat{\mathbf{u}}^{n} + \frac{\Delta t}{\rho^{n+\frac{1}{2}}} \hat{\mathbf{f}}^{n+\frac{1}{2}} - \Delta t \, \widehat{S_{h}(\mathbf{u}^{n+\frac{1}{2}})\mathbf{u}^{n+\frac{1}{2}}}\right)$$
(2.21)

From  $\nabla \cdot \mathbf{u} = \mathbf{0}$ , we have

$$\nabla \cdot \left(\frac{\rho^{n+\frac{1}{2}}}{\Delta t} - \mu \frac{L_h}{2}\right) \hat{\mathbf{u}}^{n+1} = -\Delta \hat{p}^{n+\frac{1}{2}} + \nabla \cdot \frac{\rho^{n+\frac{1}{2}}}{\Delta t} \left(\frac{\Delta t}{\rho^{n+\frac{1}{2}}} \, \hat{\mathbf{f}}^{n+\frac{1}{2}} - \Delta t \widehat{S_h(\mathbf{u}^{n+\frac{1}{2}}) \, \mathbf{u}^{n+\frac{1}{2}}}\right)$$
$$\Delta \hat{p}^{n+\frac{1}{2}} = \frac{\rho^{n+\frac{1}{2}}}{\Delta t} \left(\widehat{(rhs_u)}_x + \widehat{(rhs_v)}_y\right)$$
(2.22)

Since, from (2.8), we have

$$\Delta \hat{p} = \hat{p}_{xx} + \hat{p}_{yy}$$
  
$$\Delta \hat{p} = -\sum_{j,k} \left( \left( \frac{\sin(2\pi j \ dx)}{dx} \right)^2 + \left( \frac{\sin(2\pi k \ dy)}{dy} \right)^2 \right) e^{imh_x 2\pi j + inh_y 2\pi k} p_{j,k}$$

Then

$$\Delta \hat{p}_{j,k}^{n+\frac{1}{2}} = -\left(\left(\frac{\sin(2\pi j \ dx)}{dx}\right)^2 + \left(\frac{\sin(2\pi k \ dy)}{dy}\right)^2\right) p_{j,k}$$

And from (2.9), we have

$$(\widehat{rhs_u})_x = i \sum_{j,k} \left(\frac{\sin(2\pi j \, dx)}{dx}\right) e^{imh_x 2\pi j + inh_y 2\pi k} rhs_u$$
$$(\widehat{rhs_v})_y = i \sum_{j,k} \left(\frac{\sin(2\pi k \, dy)}{dy}\right) e^{imh_x 2\pi j + inh_y 2\pi k} rhs_v$$

Thus, from (2.22) we have that

$$-\left(\left(\frac{\sin(2\pi j \, dx)}{dx}\right)^2 + \left(\frac{\sin(2\pi k \, dy)}{dy}\right)^2\right) p_{j,k}$$
$$= \frac{\rho^{n+\frac{1}{2}}}{\Delta t} \left(i \left(\frac{\sin(2\pi j \, dx)}{dx}\right) rhs_u + i \left(\frac{\sin(2\pi k \, dy)}{dy}\right) rhs_v\right)$$

We know that

$$\hat{\mathbf{u}}^{n+1} = \begin{pmatrix} \hat{u}^{n+1} \\ \hat{v}^{n+1} \end{pmatrix} \tag{2.23}$$

Thus, from (2.21), we have  $\hat{u}^{n+1}$  as following

$$\hat{u}^{n+1} = \frac{-\nabla \hat{p} + \frac{\rho^{n+\frac{1}{2}}}{\Delta t} \widehat{rhs_{u}}}{\left(\frac{\rho^{n+\frac{1}{2}}}{\Delta t} - \mu \frac{L_{h}}{2}\right)} = \frac{\widehat{rhs_{u}} - i\left(\frac{\Delta t}{\rho^{n+\frac{1}{2}}}\right) \left(\frac{\sin(2\pi j \, dx)}{dx}\right) \hat{p}^{n+\frac{1}{2}}}{1 - \mu \frac{\Delta t}{\rho^{n+\frac{1}{2}}} \frac{L_{h}}{2}}$$
(2.24)

Similarly, for the second component  $\mathbf{u}^{n+1}$ , we have

$$\hat{v}^{n+1} = \frac{-\nabla \hat{p} + \frac{\rho^{n+\frac{1}{2}}}{\Delta t} \widehat{rhs_{v}}}{\left(\frac{\rho^{n+\frac{1}{2}}}{\Delta t} - \mu \frac{L_{h}}{2}\right)} = \frac{\widehat{rhs_{v}} - i\left(\frac{\Delta t}{\rho^{n+\frac{1}{2}}}\right) \left(\frac{\sin(2\pi k \ dy)}{dy}\right) \hat{p}^{n+\frac{1}{2}}}{1 - \mu \frac{\Delta t}{\rho^{n+\frac{1}{2}}} \frac{L_{h}}{2}}$$
(2.25)

We now find  $\frac{L_h}{2}$  in (2.24) and (2.25) From (2.7), we have

$$\hat{\mathbf{u}}_{xx}^{n+1} = \sum_{j,k} \frac{1}{h_x^2} \left( -4\sin^2\left(\frac{h_x 2\pi j}{2}\right) \right) e^{imh_x 2\pi j + inh_y 2\pi k} \mathbf{u}_{j,k}^{n+1}$$
$$= -\sum_{j,k} e^{imh_x 2\pi j + inh_y 2\pi k} \frac{\sin^2\left(\frac{h_x 2\pi j}{2}\right)}{\left(\frac{h_x}{2}\right)^2} \mathbf{u}_{j,k}^{n+1}$$

Assume  $dx = \frac{hx}{2}$  and  $dy = \frac{hy}{2}$ , then

$$\frac{L_h}{2} = -\frac{1}{2} \left( \frac{\sin(2\pi j \, dx)}{dx} \right)^2 - \frac{1}{2} \left( \frac{\sin(2\pi k \, dy)}{dy} \right)^2$$

The denominator of (2.24) and (2.25) is equal to

$$1 - \mu \frac{\Delta t}{\rho^{n+\frac{1}{2}}} \frac{L_h}{2} = 1 + 2\mu \frac{\Delta t}{\rho^{n+\frac{1}{2}}} \left( \left( \frac{\sin(2\pi j \ dx)}{dx} \right)^2 + \left( \frac{\sin(2\pi k \ dy)}{dy} \right)^2 \right)$$

According to equations (2.24) and (2.25), we can find  $\hat{u}^{n+1}$  and  $\hat{v}^{n+1}$ . Then applying the inverse Fast Fourier Transform, we could obtain the updated velocity  $\mathbf{u}^{n+1}$ , which means moving the immersed structure at the local fluid velocities thereby enforcing no slip boundary conditions. As for the inverse Fast Fourier Transform, it could be found in Trefethen's textbook [7]. Using the updated velocity to replace the old version. Then we could calculate the next position.

### 2.7 Modeling the Structure Part

Let  $\theta$  ( $0 \le \theta \le 2\pi$ ) be the parametrization of the structure, which we assume to be a closed non-self intersecting closed loop immersed in the fluid. Let  $\mathbf{F}(\theta, t)$  be the force acting on the filament. Then  $\mathbf{f}$ , the force acting on the fluid in the Navier-Stokes equation, is given by

$$\mathbf{f}(x,t) = \int_0^{2\pi} \mathbf{F}(\theta,t) \delta\Big(x - X(\theta,t)\Big) d\theta$$
(2.26)

and the movement of the filament is governed by the equation

$$\frac{\partial \mathbf{X}}{\partial t} = \mathbf{u} \Big( \mathbf{X}(\theta, t), t \Big)$$
(2.27)

$$= \int_{\Omega} \mathbf{u}(x,t) \,\delta\Big(\mathbf{x} - \mathbf{X}(\theta,t)\Big) d\mathbf{x}$$
(2.28)

where  $\Omega = [0, L] \times [0, L]$  is the domain of the fluid and  $\delta(\mathbf{x}) = \phi(x_1)\phi(x_2)$ , where  $\mathbf{x} = (x_1, x_2)$ . In the above,  $\phi(r)$  is continuous for all r and  $\phi(r) = 0$  for  $|r| \ge 2$ . Define

$$\begin{split} \phi(r) &= 0, \quad r \le 2\\ &= \frac{1}{8} \Big( 5 + 2r - \sqrt{-7 - 12r - 4r^2} \Big), \quad -2 \le r \le -1\\ &= \frac{1}{8} \Big( 3 + 2r + \sqrt{1 - 4r - 4r^2} \Big), \quad -1 \le r \le 0\\ &= \frac{1}{8} \Big( 3 - 2r + \sqrt{1 + 4r - 4r^2} \Big), \quad 0 \le r \le 1\\ &= \frac{1}{8} \Big( 5 - 2r - \sqrt{-7 + 12r - 4r^2} \Big), \quad 1 \le r \le 2\\ &= 0, \qquad r \ge 2 \end{split}$$

# Chapter 3

### **Active and Passive Forces**

In the section, we would like to describe the fiber models implemented in this thesis. As for the Navier-Stokes Equation, it is affected by two kinds of forces. One of the force is the internal force which includes pressure and viscosity. The other is external forces. In this chapter, all kind of forces are the external force in the fluid-structure interaction. Various fiber models give the immersed boundary certain desirable material properties relevant to many scientific applications. We should composite the total fiber model to obtain the entirely force in Navier-Stokes Equations.

In this thesis the following types of fiber models are implemented:

- 1. Peskin's Model
- 2. Spring Model
- 3. Torsional Spring(Beam) Model
- 4. Muscle-Fluid-Structure Model
- 5. 3-Element-Hill Model

Once the deformation energy has been calculated in the algorithm, we get

$$E(X(r,t),t) = \sum_{k=0}^{m} E_k(X_{k,1}, X_{k,2}, \dots, X_{k,N_k})$$
(3.1)

the corresponding elastic forces can be computed via derivatives of the elastic energy, where the elastic deformation force at point c of fiber model k is calculated as

$$F_{k,c}(X(r,t),t) = -\frac{\partial E(X(r,t),t)}{\partial X_{k,c}}$$
(3.2)

Note that X contains the coordinates of all immersed boundary points, M means the number of fiber structures in the system,  $N_k$  means the number of immersed boundary points in fiber structure M, and the negative sign is chosen to drive the system towards a minimal energy state. In addition, (3.1) described the compositing of the deformation energy from all respective fiber models. [12]

#### 3.1 Peskin's Model

According to Peskin's Model[3], the force applied by arc  $d\theta$  if immersed boundary to fluid. Generally, Force balance in interval (a, b) yields. Let

$$T(\theta, t) =$$
tension in immersed boundary (3.3)

$$\tau = \frac{\partial \mathbf{X}/\partial \theta}{|\partial \mathbf{X}/\partial \theta|} = \text{unit tangent to immersed boundary}$$
(3.4)

$$\mathbf{F}(\theta, t)d\theta$$
 = force applied by arc  $d\theta$  if immersed boundary to fluid (3.5)

$$-\int_{a}^{b} \mathbf{F} d\theta = \text{force of fluid on boundary}$$
(3.6)

$$0 = T\tau|_{\mathbf{a}}^{\mathbf{b}} - \int_{\mathbf{a}}^{\mathbf{b}} \mathbf{F} d\theta \qquad (3.7)$$

$$= \int_{a}^{b} \left( \frac{\partial}{\partial \theta} (T\tau) - \mathbf{F} \right) \mathbf{d}\theta \tag{3.8}$$

(3.9)

Since a, b are arbitrary

$$\mathbf{F} = \frac{\partial}{\partial \theta} (T\tau) \tag{3.10}$$

$$= \frac{\partial T}{\partial \theta} \tau + \mathbf{T} \frac{\partial \tau}{\partial \theta}$$
(3.11)

$$= \frac{\partial T}{\partial \theta} \tau + \mathbf{T} \mathbf{Cn}$$
(3.12)

where C = curvature, and  $\mathbf{n} =$  unit normal to boundary. In generally, T is some function of  $\left|\frac{\partial \mathbf{X}}{\partial \theta}\right|$ The special case is

$$T = K \left| \frac{\partial \mathbf{X}}{\partial \theta} \right| \tag{3.13}$$

Therefore,

$$T\tau = K \left| \frac{\partial \mathbf{X}}{\partial \theta} \right| \frac{\partial \mathbf{X} / \partial \theta}{|\partial \mathbf{X} / \partial \theta|} = K \frac{\partial \mathbf{X}}{\partial \theta} = K \frac{\partial \mathbf{X}}{\partial \theta}$$
(3.14)

So

$$F = \frac{\partial}{\partial \theta} (T\tau) = K \frac{\partial^2 \mathbf{X}}{\partial \theta^2}$$
(3.15)

#### 3.2 Spring Model

The following model is cited the Battista's paper directly [12]. Springs is a kind of force that resistance to stretching between each consecutive Lagrangian points(nodes) can be achieved by modeling the connections with Hookean springs of resting length  $R_L$  and spring constant  $k_s$ . Thus, referring to the Hookean springs, it related to Hooke's law which is a law of physics that states that that the force needed to extend or compress a spring by some distance x cales linearly with respect to that distance. That is:  $F_s = kx$ , where k is a constant factor characteristic of springs: its stiffness, and x is small compared to the total possible deformation of the spring. Since we need to consider the resting the length of springs, then we obtain the energy formula as following:

$$E_{spring} = \frac{1}{2} (||X_L - X_R|| - R_L)^2$$
(3.16)

where  $X_L$  and  $X_R$  are left and right node coordinates respectively. Applying for (3.2), we have the deformation force is given by a derivative on the elastic energy:

$$F_{spring} = k_x \left( 1 - \frac{R_L}{||X_L - X_R||} \right) \cdot \begin{pmatrix} x_L - x_R \\ y_L - y_R \end{pmatrix}$$
(3.17)

As you can see in the Figure 3.1, when the resting length is exactly equal to  $R_L$ , there is no force between two consecutive nodes. While the length greater or less than  $R_L$ , the spring provides forces for two consecutive nodes.

#### 3.3 Torsional Spring Model

The following model is cited the Battista's paper directly [12]. Torsional springs is also called as beams, which means resistance to bending three consecutive Lagrangian points (nodes). The model assumes a desired angle  $\theta$ , a prescribed 'curavture' between the three Lagrangian poins, with corresponding bending stiffness  $k_B$ . Then the bending energy is given as

$$E_{bend} = \frac{1}{2} k_B (\hat{z} \cdot (X_R - X_M) \times (X_M - X_L) - C)^2$$
(3.18)





where  $X_R$ ,  $X_M$  and  $X_L$  are right, middle and left nodes, and  $C = d_{LM} d_{MR} \sin \theta$ . Note that C is not the standard definition of curvature, but a curvature defined at the desired angle  $\theta$  and distances between links,  $d_{LM}$  and  $d_{MR}$ .

The penalty force is designed to drive any deviations in the angle between these links back towards a lower energy state. The corresponding bending force is given by

$$F_{bend} = k_B \Big( (x_R - x_M)(y_M - y_L) - (y_R - y_M)(x_M - x_L) - C \Big) \cdot \begin{pmatrix} (y_M - y_L) + (y_R - y_M) \\ -(x_R - x_M) - (x_M - x_L) \end{pmatrix}$$
(3.19)

As you can see in the Figure 3.2, there is no force when the angle of the three consecutive nodes is  $\theta$ . Otherwise, the torsional spring provides the force for middle point of the three consecutive nodes.

#### 3.4 Muscle-Fluid-Structure Model

The following model is cited the Battista's paper directly [12]. According to the Model by N.A Battista [12]. For his model, it described the muscle model combined with a forcevelocity and length-tension relationship in muscle without coupling in the underlying cellular processes like calcium signaling, myosin cross-bridge attachment and detachment, or filament compliance.

The muscle force is influenced by the speed of muscle contraction. Traditionally, a Hill

Figure 3.2: Torsional Spring Force



model is used to describe this relationship and takes the following form [1, 14].

$$V_F = \frac{b(F_{max} - F)}{F + a} \tag{3.20}$$

where  $V_F$  is the muscle fiber's shortening velocity, F is the force generated by the fiber, and  $F_{max}$  is the maximum load at zero contractile velocity. a and b are the parameters which are related to the internal thermodynamics of muscle.

The muscle force is also related to a function of its length. Initially when the thick filaments begin to bind to the thin filaments, the resulting force increases as the muscle shortens. However, if the muscle is contracted too far, there are fewer myosin heads to attach to the actin filaments, and the resulting force exerted is smaller. Therefore, the maximal muscle tension is located between the two extremes. A simple model of a length-tension relationship as following [4],

$$F_l = F_{l0} \exp\left[-\left(\frac{Q-1}{SK}\right)^2\right]$$
(3.21)

where  $Q = \frac{L_F}{L_{F0}}$  is the ratio of the length of the muscle fibers to the length when they reach the maximum tension,  $F_l$  is the maximum isometric tension at a given fiber length  $L_F$ ,  $F_{l0}$ is the maximum isometric force when they reach the optimum length of the muscle fibers, and SK is a parameter specific for each muscle.

Follow by Battista's model, combining (3.20) and (3.21) is to take product of their normalized versions, as in [12, 5]. The model as following

$$F_{muscle}(L_F, V_F) = a_f F_{max} F_1(L_F) F_2(V_F)$$

$$(3.22)$$

where  $a_f$  is the activation strength of the muscle related to the Calcium and  $F_{max}$  is the normalized maximum isometric force at the full activation of the muscle fibers at their optimum lengths, and  $F_1(L_F)$  and  $F_2(V_F)$  are normalized versions of (3.20) and (3.21), given by

$$F_1(L_F) = \exp\left[-\left(\frac{L_F/L_{F0}-1}{SK}\right)^2\right]$$
 (3.23)

$$F_2(V_F) = \frac{1}{F_{max}} \left[ \frac{bF_{max} - aV_F}{b + V_F} \right]$$
(3.24)

Since  $a_f$  is the activation strength of the muscle related to the Calcium Dynamics, we used the Shannon Model as the activation strength and re-scale the range from 0 to 1 [13].

Figure 3.3: Force-velocity and length-tension curves illustrating the respectively



As you can see in the Figure 3.3, the  $F_2$  decreases with the increasing of velocity and  $F_muscle$  related to the actin and myosin cross-bridges at the sarcomere level.

#### 3.5 3-Element-Hill Model

The following model is cited the Battista's paper directly [12]. The 3-Element Hill model activation describes sustained muscle contraction by modeling the actin and myosin crossbridges, muscle tendon, and connective tissues for a muscle. The model has a contractile element which models the force generated by the actin and myosin cross-bridges at the sarcomere level, and two non-linear spring elements, and one of them is parellel element and the other one is the series element with contractile element. The series element models related to the muscle tendon, and has a soft tissue response and provides energy storing mechanism. The parallel element is responsible for the passive behavior when the muscle is stretched. For the contractile element, it is the same model we described in the above section 3.4.

Let  $F_{CE}$ ,  $F_{SE}$  and  $F_{PE}$  be contractile, series, and parallel elements respectively. In mechanics, two or more springs share the same force in series if they are connected end-toend. If two or more springs in parallel which means they are connected side-by-side, the force of this case is summation of the force of each side. Thus, we have the formula as following

$$F_{tot} = F_{SE} + F_{PE} \tag{3.25}$$

$$F_{CE} = F_{SE} \tag{3.26}$$

where  $F_{tot}$  is the total force produced by muscle contraction. For the muscle shortening, we have

$$L_{tot} = L_{CE} + L_{SE} = L_{PE} \tag{3.27}$$

where  $L_{tot}$  is the total length of the muscle.

The contractile element is similar with the above section. Thus, we can assume it related to the length-tension and force-velocity relationship of muscle. By equation (3.22), we have

$$F_{CE} = a_f F_{max} F_1(L_{CE}) F_2(V_{CE})$$
(3.28)

where  $L_{CE}$  is the length of the muscle fibers and  $V_{CE}$  is the contraction speed of the muscle fibers.

From Figure 3.4, this is the picture of the three-element Hill model.





# Chapter 4

# **Numerical Simulation**

In this chapter, we modified the model which comes from Charles S. Peskin [2]. This thesis shows some numerical simulations with tables and results of three cases of different combinations of the following 4 models.

- 1. Original Peskin's Model
- 2. Spring Model
- 3. Torsional Spring Model
- 4. 3-Element-Hills Model

The combination of the 4 models is based on

$$F = \omega_0 F_{peskin} + \omega_1 F_{spring} + \omega_2 F_{torsional spring} + \omega_3 F_{muscle}$$

$$\tag{4.1}$$

where  $\omega_i$  is the weight of each force. The initial velocity **u** is set to zero. The initial geometry of the structure is a circle centered at  $(\frac{L}{2}, \frac{L}{2})$  with radius  $\frac{L}{4}$ .  $[0, L] \times [0, L]$  is the size of the domain.

Firstly, we provide the proper parameters in the following table.

### 4.1 Parameters

Parameter Values and Their Units						
Measurement	Symbol	Magnitude	Unit			
Length	L	1.0	mm			
Number of Grid	N	64	Each			
Length of each Grid	h	$\left  \frac{1}{64} \right $	mm			
Number of Points	Nb	202	Each			
Density	$\rho$	1.06	$kg/m^2$			
Dynamic Viscosity	$\mid \mu$	0.04	$N * s/m^2$			
Time-step	dt	0.01	second			
Degree	$\alpha$	1	No unit			
x-Direction Length	Lx	1.0	mm			
y-Direction Length	Ly	1.0	mm			
Rest of Length	Lr	0.015	mm			
Stiffness	$K_{\rm spring}$	300	N/mm			
Degree	$\alpha$	1	No unit			
x-Direction Length	Lx	1.0	mm			
y-Direction Length	Ly	1.0	mm			
Rest of Length	C	0	mm			
Stiffness	$K_{\mathrm{beam}}$	150	N/mm			
Degree	$\alpha$	1	No Unit			
x-Direction Length	Lx	1.0	mm			
y-Direction Length	Ly	1.0	mm			
Stiffness	$K_{\rm beam}$	5	N/m			
Maximum Tension	$F_{max}$	212	KPa			
Hill parameter a	a	0.25	No Unit			
Hill parameter b	b	4	No Unit			
Muscle constant	Sk	0.3	No Unit			
Length for max muscle ten-	LF0	0.018	mm			
sion						

#### 4.2 Case 1

In the first case, we show the simulation which only contains Original Peskin's model and 3-Element Hill model. Thus, suppose  $\omega_0 = 1$  and  $\omega_3 = 1$ , and set  $\omega_1$  and  $\omega_2$  to 0,



Figure 4.1: Case 1:  $\omega_0 = 1, \omega_1 = 0, \omega_2 = 0$  and  $\omega_3 = 1$ .

The above Figure (4.1) show the simulation at t = 1.25, 2.25, 2.75, and 4 respectively. Since this case does not contain the spring and torsional spring model. The nodes apparently converges to the diagonal at t = 4. We will add spring and torsional spring model into the second case, and observe the changes.

#### 4.3 Case 2

In this case, we keep the Original Peskin's model and 3-Element Hill model, and add spring and torsional spring model into the case. Thus, let  $\omega_i = 1$ , for i = 0, 1, 2, 3.



Figure 4.2: Case 2:  $\omega_0 = 1, \omega_1 = 1, \omega_2 = 1$  and  $\omega_3 = 1$ .

The above Figure (4.2) show the simulation at t = 1.25, 2.25, 3.25, and 4 respectively. A comparison of Figure (4.2) and Figure (4.1), it is obvious that the force is more evenly spread between the nodes in Case 2. However, the shape of the structure is not convex enough. Since the original Peskin's model provides the curvature force to the shape, we enlarge the weight of the original Peskin's model in the next case.

#### 4.4 Case 3

In this case, we keep the same weight of spring, torsional spring and 3-Element Hill model as Case 2, and increase the weight of the Original Peskin's model. Let  $\omega_0 = 1.5$ 



Figure 4.3: Case 3:  $\omega_0 = 1.5, \omega_1 = 1, \omega_2 = 1$  and  $\omega_3 = 1$ .

The above Figure (4.3) show the simulation at t = 1.25, 2.25, 3.5, and 4 respectively. According to Figure (4.1), Figure (4.2) and Figure (4.3), after increasing of weight of the original Peskin's model, the structure begins doing the periodic movement and the shape and force magnitude of each time step is stable.

# Chapter 5

# Conclusion

In this thesis, we modified the numerical results of Charles Peskin in [2] to simulate the active and passive forces in the Fluid-Structure Interaction problem of the human heart. In addition to Peskin model, we implemented three other models including the spring model, the torsional spring (or beam) model, and the 3-element Hill model into our simulations. Using Fast Fourier Transform (FFT), we solved the Navier-Stokes equation. FFT allows computational time to be  $O(n \log n)$  instead of  $O(n^2)$ . For numerical implementations, the domain is taken to be  $[0,1] \times [0,1]$  and we used the parameter values from Battista's paper [12]. In Chapter 1, we introduced the left ventricle and provided background of the Navier-Stokes equations. In Chapter 2, we described the half-step method to solve the Navier-Stokes equation, and explained the Fast Fourier Transform method [7]. At the end of Chapter 2, we illustrated how to update the velocity  $\mathbf{u}(x,t)$  in the Fluid-Structure Interaction Problem. In Chapter 3, we explained details of the above mentioned four models which come from Peskin's Note and Battista's paper [2, 12]. Chapter 4 is numerical results. We implemented three combinations of the model mentioned in Chapter 3. Since to simulate the movement of the left ventricle, we added the Calcium Dynamics from the Shannon model [13]. The best result was obtained if we weighted Peskin model 1.5 times the other three models. Because of the lack of time, we did not find the optimal combination of these four forces to produce simulation that best resembles a beating heart.

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# Appendix A

# Matlab Code

Code for all Matlab functions used in simulations is listed below.

```
1 % ib2D.m
2 % This script is the main program.
3 clear all
4 close all
5
6 global dt Nb N h rho mu ip im a;
7 global kp km dtheta K LF0 dt RL;
8 initialize
9 init_a
10
11 x_master = X(kp, 1);
12 x_{slave} = X(km, 1);
13 y_master = X(kp, 2);
14 y_slave = X(km, 2);
15
16 for i = 1:202
      dx = x_master(i) - x_slave(i);
17
       dy = y_master(i) - y_slave(i);
18
       RL(i) = sqrt(dx^{2} + dy^{2});
19
20 end
^{21}
_{22} for i = 1:202
       dx = x_master(i) - x_slave(i);
23
       dy = y_master(i) - y_slave(i);
24
       LFO(i) = 0.018;
25
26 end
27
28 tmp = load('shannonresult');
29 ti = tmp(:,1); % time
30 Cai = tmp(:,2); % calcium
31 X_P = X;
32
```

```
33
34 for clock=1:clockmax
    XX=X+(dt/2)*interp(u,X);
35
     af = Cai(clock*10);
36
     ff=spread(Force(XX, X_P, af), XX);
37
     [u,uu]=fluid(u,ff);
38
39
     X_PP = X_P;
     X_P = X;
40
     X=X+dt*interp(uu,XX);
^{41}
42
     %animation:
43
44
     vorticity=(u(ip,:,2)-u(im,:,2)-u(:,ip,1)+u(:,im,1))/(2*h);
     contour(xgrid, ygrid, vorticity, values)
45
     hold on
46
     plot(X(:,1),X(:,2),'ko')
47
     axis([0, L, 0, L])
48
     caxis(valminmax)
49
50
     axis equal
51
     axis manual
52
    drawnow
53
    hold off
54
55 end
```

```
1 %initialize.m
2 L=1.0
3 N=64
_{4} h=L/N
5 ip=[(2:N),1]
6 im=[N,(1:(N-1))]
7 Nb=ceil(pi*(L/2)/(h/2))
8 dtheta=2*pi/Nb
9 kp=[(2:Nb),1]
10 km=[Nb, (1:(Nb-1))]
11 K=1
12 rho=1.06
13 mu=0.04
14 tmax=4;
15 dt=0.01
16 clockmax=ceil(tmax/dt)
17
18 for k=0:(Nb-1)
   theta=k*dtheta;
19
    X(k+1, 1) = (L/2) + (L/4) * \cos(theta);
20
    X(k+1,2) = (L/2) + (L/4) * sin(theta);
^{21}
22
   end
23
24 u=zeros(N,N,2);
25 for j1=0:(N-1)
    x=j1*h;
26
    u(j1+1,:,2) = eps*cos(2*pi*x/L);
27
  end
28
29
30 vorticity=(u(ip,:,2)-u(im,:,2)-u(:,ip,1)+u(:,im,1))/(2*h);
31 dvorticity=(max(max(vorticity))-min(min(vorticity)))/5;
32 values= (-10*dvorticity):dvorticity:(10*dvorticity);
33 valminmax=[min(values), max(values)];
34 xgrid=zeros(N,N);
35 ygrid=zeros(N,N);
36 for j=0:(N-1)
     xgrid(j+1,:)=j*h;
37
     ygrid(:,j+1)=j*h;
38
39 end
40
41 set(gcf, 'double', 'on')
42 contour (xgrid, ygrid, vorticity, values)
43 hold on
44 plot(X(:,1),X(:,2), 'ko')
45 axis([0,L,0,L])
46 caxis (valminmax)
47 axis equal
48 axis manual
49 drawnow
50 hold off
```

```
1 function F=Force(X, X_P, af)
2 global kp km dtheta K LF0 dt Nb RL;
3
4
5
6 F=K*(X(kp,:)+X(km,:)-2*X)/(dtheta*dtheta);
7
8 %F_peskin = mean(F) % ? 20191201-WL;
9
10 k_Spring = 300; % 300;
11 % F_spring1 = 1/2*k_Spring*(1 - (X(kp,:)+X(km,:))/norm(X(kp,:)-X(km,:)));
12 % F_spring2 = X(kp,:)-X(km,:);
13 % F_spring = F_spring1.*F_spring2;
14 💡
15 % F = F + F_{spring};
16
17 kk=1:Nb;
18 x_master = X(kp, 1);
19 x_slave = X(kk,1);
_{20} y_master = X(kp,2);
21 y_slave = X(kk,2);
               % 20191201-WL;
22 % Lx = 1;
23 % Ly = 1;
24 % L_r = 2*pi*Lx/4/202;
25 alpha = 1;
26
                                      % Initialize storage for x-forces
27 \text{ fx} = \text{zeros}(Nb, 1);
_{28} fy = fx;
                                      % Initialize storage for y-forces
29
_{30} for i = 1:Nb
31 dx = x_master(i) - x_slave(i);
32 dy = y_master(i) - y_slave(i);
33
34 %
        if abs(dx) > Lx/2
                                                  % 20191201-WL;
35 %
             dx = sign(dx) * (Lx - sign(dx) * dx);
36   😽
        end
  8
37
if abs(dy) > Ly/2
dy = sign(dy) * (Ly - sign(dy) * dy);
40 %
        end
41
42 sF_x = 0.5*(alpha+1) * k_Spring * ( sqrt( dx^2 + dy^2 ) - RL(i) )^(alpha) * (
      dx / sqrt(dx^2+dy^2));
  sF_y = 0.5*(alpha+1) * k_Spring * (sqrt(dx^2 + dy^2) - RL(i))^(alpha) * (
43
      dy / sqrt(dx^2+dy^2) );
44
  % THE FORCE IS THE ONE APPLIED ON SPRING TO HAVE THIS DEFORMATION; 20191201-WL
45
   f_x(kp(i),1) = f_x(kp(i),1) + sF_x; % Sum total forces for node, i in x-
46
       direction (this is MASTER node for this spring)
   fy(kp(i),1) = fy(kp(i),1) + sF_y; % Sum total forces for node, i in y-
47
```

```
direction (this is MASTER node for this spring)
48
                                          % Sum total forces for node, i in x-
    fx(kk(i), 1) = fx(kk(i), 1) - sF_x;
49
       direction (this is SLAVE node for this spring)
    fy(kk(i),1) = fy(kk(i),1) - sF_y; % Sum total forces for node, i in y-
50
       direction (this is SLAVE node for this spring)
51
  end
52
53
54 \text{ F_spring} = [fx fy];
55
  % F_spring_force = mean(F_spring)
56
57
58
59 %new, beam
60 kk=1:Nb;
k_{Beam} = 150;
62 \ \% \ C = 0;
63 % C=LF0(1)*LF0(2)*sin(dtheta);
64
65 fx = zeros(Nb, 1);
                                       % Initialize storage for x-forces
                                        % Initialize storage for y-forces
66
  fy = fx;
67
68
69 x_left = X(km, 1);
70 x_middle = X(kk, 1);
71 x_right = X(kp, 1);
72
y_{-} = X(km, 2);
_{74} y_middle = X(kk, 2);
_{75} y_right = X(kp, 2);
76
77
  for i=1:Nb
                                   % xPt of 1ST Node Pt. in beam
       X_L = x_left(i);
78
       X_M = x_middle(i);
                                    % xPt of 2ND (MIDDLE) Node Pt. in beam
79
       X_R = x_right(i);
                                   % xPt of 3RD Node Pt. in beam
80
81
       Y_L = y_left(i);
                                   % yPt of 1ST Node Pt. in beam
82
       Y_M = y_middle(i);
                                     % yPt of 2ND (MIDDLE) Node Pt. in beam
83
                                    % yPt of 3RD Node Pt. in beam
       Y_R = y_right(i);
84
85
           % Compute Cross-Product
86
       cross_prod = (X_R-X_M) * (Y_M-Y_L) - (Y_R-Y_M) * (X_M-X_L);
87
88
       C= sqrt( (X_M - X_L)<sup>2</sup> + (Y_M - Y_L)<sup>2</sup>) * sqrt( (X_M - X_R)<sup>2</sup> + (Y_M - Y_R
89
          )^2 )*sin(dtheta);
90
       % FORCES FOR LEFT NODE
91
       bF_x_L = -k_Beam * (cross_prod - C) * (Y_R-Y_M);
92
       bF_y_L = k_Beam * (cross_prod - C) * (X_R-X_M);
93
94
```

```
% FORCES FOR MIDDLE NODE
95
       bF_xM = kBeam * (cross_prod - C) * ((Y_M-Y_L) + (Y_R-Y_M));
96
       bF_yM = -k_Beam * (cross_prod - C) * ((X_R-X_M) + (X_M-X_L));
97
98
       % FORCES FOR RIGHT NODE
99
       bF_xR = -k_Beam * (cross_prod - C) * (Y_M-Y_L);
100
101
       bF_y_R = k_Beam * (cross_prod - C) * (X_M-X_L);
102
       f_x(km(i),1) = f_x(km(i),1,1) - bF_xL; % Sum total forces for left node,
103
           in x-direction (this is LEFT node for this beam)
       fy(km(i),1) = fy(km(i),1,1) - bF_y_L; % Sum total forces for left node,
104
           in y-direction (this is LEFT node for this beam)
105
       fx(kk(i),1) = fx(kk(i),1,1) + bF_x_M; % Sum total forces for middle node,
106
            in x-direction (this is MIDDLE node for this beam)
       fy(kk(i),1) = fy(kk(i),1,1) + bF_y_M; % Sum total forces for middle node,
107
            in y-direction (this is MIDDLE node for this beam)
108
       fx(kp(i),1) = fx(kp(i),1,1) - bF_x_R; % Sum total forces for right node,
109
           in x-direction (this is RIGHT node for this beam)
       fy(kp(i),1) = fy(kp(i),1,1) - bF_y_R; % Sum total forces for right node,
110
           in y-direction (this is RIGHT node for this beam)
111
112
   end
113
   F_beam = [fx fy];
114
115
   % F_beam_force = mean(F_beam)
116
117
118
119 %Muscle Mode
120 FMax = 212;
121 SK=0.3;
122 a=0.25;
123 b=4;
124 x_master_P = X_P(kp,1); % x_master_P = X_P(kp,1);
                            % x_slave_P = X_P(km,1);
   x_slave_P = X_P(kk, 1);
125
126 y_master_P = X_P(kp, 2); % y_master_P = X_P(kp, 2);
127 \text{ y_slave_P} = X_P(kk, 2);
                            % y_slave_P = X_P(km, 2);
128 kSpr = 5;
129
  fx = zeros(Nb, 1);
                                       % Initialize storage for x-forces
130
   fy = fx;
                                       % Initialize storage for y-forces
131
132
   for i= 1:Nb
133
       dx = x_{master}(i) - x_{slave}(i);
134
       dy = y_master(i) - y_slave(i);
135
       LF = sqrt(dx^2 + dy^2);
136
       Q=LF/LF0(i);
137
       F1 = \exp(-((Q-1)/SK)^2);
138
       P0 = FMax * F1;
139
```

```
140
        dx_P = x_master_P(i) - x_slave_P(i);
        dy_P = y_master_P(i) - y_slave_P(i);
141
       LF_P = sqrt(dx_P^2 + dy_P^2);
142
       v = abs(LF-LF_P)/(dt/2);
143
       F2 = (1/P0) * (b*P0-a*v) / (v+b);
144
       Fm = af*FMax*abs(F1*F2);
145
       mF_x = Fm \star (dx/LF);
146
       mF_y = Fm \star (dy/LF);
147
148
149
        % PE force
150
        sF_PE_x = kSpr * (sqrt(dx^2 + dy^2) - LF0(i)) * (dx / sqrt(dx^2+dy^2))
151
            );
        sF_PE_y = kSpr * (sqrt(dx^2 + dy^2) - LF0(i)) * (dy / sqrt(dx^2+dy^2))
152
            );
153
154
        fx(kp(i), 1) = fx(kp(i), 1) + mF_x+sF_PE_x;
155
       fy(kp(i), 1) = fy(kp(i), 1) + mF_y+sF_PE_y;
156
157
       fx(kk(i), 1) = fx(kk(i), 1) - mF_x - sF_PE_x;
158
159
       fy(kk(i), 1) = fy(kk(i), 1) - mF_y - sF_PE_y;
160
161
   end
162
163
   F_Muscle = [fx fy];
164
165
   % F_Muscle_force = mean(F_Muscle)
166
167
168
   % %Curvature Force, LX is wrong;
169
   % k_curve = 1e-08;
170
171 % x_deri = (X(kp,1) - X(kk,1))/(dtheta);
                                                            %first deri for x-
      direction
172 % y_deri = (X(kp,2) - X(kk,2))/(dtheta);
                                                            %first deri for y-
       direction
173 % % x_deri =
                 (X(kk,1) - X_P(kk,1))/(dtheta);
                                                                %first deri for x-
      direction
174 % % y_deri =
                  (X(kk,2) - X_P(kk,2))/(dtheta);
                                                                %first deri for y-
      direction
176 % x_second_deri = (X(kp,1) - 2*X(kk,1) + X(km,1))/(dtheta^2);
       %second deri for x-direction
177 % y_second_deri = (X(kp,2) - 2*X(kk,2) + X(km,2))/(dtheta^2);
       %second deri for y-direction
178 % % x_second_deri = (X(kk,1) - 2*X_P(kk,1) + X_PP(kk,1))/((dtheta)^2);
                      %second deri for x-direction
179 % % y_second_deri = (X(kk,2) - 2*X_P(kk,2) + X_PP(kk,2))/((dtheta)^2);
                       %second deri for y-direction
180  😵
```

```
37
```

```
181 % fx = zeros(Nb, 1);
                                       % Initialize storage for x-forces
                                       % Initialize storage for y-forces
182 % fy = fx;
183  😵
184 % for i = 1:Nb
185 % k_curvature = (x_deri(i) *y_second_deri(i) - x_second_deri(i) *y_deri(i))
      /((x_deri(i)^2 + y_deri(i)^2)^(3/2));
      N_x = -y_deri(i)/(x_deri(i)^2 + y_deri(i)^2)^0.5;
186
   00
N_y = x_deri(i)/(x_deri(i)^2 + y_deri(i)^2)^0.5;
188 💡
        fx(kk(i),1) = k_curve*k_curvature*N_x;
        fy(kk(i),1) = k_curve*k_curvature*N_y;
190  😵
        if isnan(fx(kk(i),1))
191 💡
             fx(kk(i), 1) = 0;
192 🔗
       end
if isnan(fy(kk(i),1))
194   😽
             fy(kk(i), 1) = 0;
195 %
       end
196 % end
198 % F_curvature = [fx fy];
199
200 % F_cur_force = mean(F_curvature)
201
202
203 F = F + F_Muscle; % F + F_spring + F_beam + + F_curvature;
_{204} %F = 1.5*F + F_spring + F_beam + F_Muscle;
```